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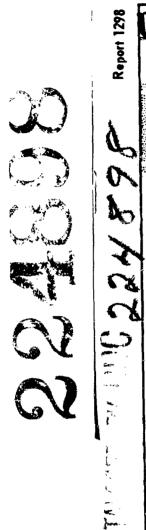
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ENVIRONMENTAL CONDITIONS OF SHIP MOTIONS AND VIBRATIONS FOR DESIGN OF RADAR SYSTEMS ON DESTROYERS AND AIRCRAFT CARRIERS

Ьу

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STRUCTURAL MECHANICS LABORATORY
RESEARCH AND DEVELOPMENT REPORT

July 1959

Report 1298

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Report 1298 NE 050-500

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NOTATION

A	Single amplitude of vibration
d	Distance from center of gravity to base of radar mast
e	Base of Napierian logarithms
E	Mean value of x^2
E _m	Mean value of x^2 for most severe condition
1	Value for the risk taken
h	Height of radar mast above neutral axis of ship
1	Mass moment of inertia of radar unit about axes fixed in its center of gravity
M	Mass of radar unit
N	Number of variations in a sample
n	Whipping frequencies
p(x)	dP/dx, Relative probability of encountering a value of magnitude x
$P\left(\boldsymbol{x}_{i}\right)$	Fraction of all members of a sample with values $x < x_i$
t	Time
æ	Magnitude of a variable
$\boldsymbol{x_i}$	A given value of z
z _m	Value of x which will be exceeded at least once in a fraction, f , of a great number of similar tests
y	Function of the risk
ψ	Roll angle
ϕ_p	Pitch angle
ω,	Roll frequency
ω _p	Pitch frequency

ABSTRACT

From rigid-body motions and environmental vibrations measured on aircraft carriers and destroyers over a wide range of service conditions, environmental conditions of vibration and ship motions are defined for use in the design of radar installations. Statistical distribution patterns of motions applicable to ship operation over a long period and those for the most adverse operating conditions experienced in a severe sea are used to predict extreme values.

Maximum linear accelerations in the longitudinal, athwartship, and vertical directions and maximum angular accelerations at the radar platform are estimated.

Applications of the data to design problems are discussed.

INTRODUCTION

The design and operation of search and height-finding radar have long been hampered by lack of information concerning the motions and accelerations to which this equipment is subjected. Accordingly, in January 1952 the Bureau of Ships 1 requested the David Taylor Model Basin to conduct tests on various types of ships to determine design conditions, with respect to ship's rigid-body motions, for stabilized radar installations. In October 1955² the problem was broadened to include vibratory motions associated with the elastic response of the structure as well, since successful aircraft intercept had become extremely sensitive to elevation errors in height-finding radar. Ship flexure between the radar antenna and the reference gyro equipment, which is usually remote from the antenna, and environmental vibrations at the actual antenna location can be sources of error.

Data have been collected to determine the rigid-body motions and environmental vibrations at the service platforms on radar masts, particularly on carriers and destroyers. This report evaluates the existing data on carriers and destroyers in a manner which, it is hoped, will be useful to the designer of radar installations.

The analysis of existing data is given in three parts: evaluation of ship motion data, evaluation of vibration data, and application to design.

RIGID-BODY MOTION OF SHIPS

Motion data necessary to specify structural design requirements are given in terms of maximum acceleration; and maximum pitch and roll angles to be expected over the operational life of the ship. Operational requirements are given in terms of the probability of exceeding or not exceeding given angles of roll and pitch for the most severe operating condition

^{*}References are listed on page 27.

expected. In this report ship motions are given in terms of their peak-to-peak variation; i.e., a roll angle of 30 deg will mean that the variation of roll angle is 30 deg from port to starboard.

A brief background of statistical analysis needed for the evaluation is outlined in the next section.

STATISTICAL BACKGROUND

The ship motions experienced under a given set of conditions can be described in terms of their distribution functions. It has been shown³ that the applicable distribution functions are approximated by the Rayleigh distribution for a given set of steady operating conditions such as sea state, ship speed, and heading (short-term), and by log-normal distributions if the operating conditions are allowed to vary over a wide range such as would occur over a typical year (long-term).

The Rayleigh distribution of a variable x is defined by the single parameter E, the mean square value of x; i.e., $E = x^2$. The log-normal distribution of x is defined by two parameters—the mean value of $\log x$ and the variance of $\log x$. Both the Rayleigh (Figure 5) and log-normal distributions (Figures 2,3,4,6,7, and 8) can be represented by straight lines when plotted on special graph paper. The statistical methods utilized here are discussed in References 3 and 4.

For illustrative purposes, consider one of the variables—for example, pitch angle. All pitch angles, peak to peak, are considered to be members of a statistical population. The distribution function indicates the relative probability p(x) of encountering a pitch angle of the magnitude x. Figure 1 illustrates this distribution function. The area under the curve of Figure 1 up to a value x_i is the fraction P of all members of the population which have values less than x_i . Therefore the probability of exceeding the value x_i is 100 (1—P) percent. For the Rayleigh distribution $P(x) = 1 - e^{-x^2/E}$.

The distribution patterns give the probability of exceeding any given magnitude of motion and can be utilized indesigning for endurance strength. For any set of operating conditions, characteristic and extreme values can be predicted from a knowledge of the corresponding value of E. Useful statistical estimates are made as follows:⁵

- a. The most frequent magnitude of variations is $0.707\sqrt{E}$.
- b. The average magnitude of variations is $0.866\sqrt{E}$.
- c. The most probable extreme value x_m experienced in a sample of N variations is $x_m \approx \text{constant}\sqrt{E}$. For large values of N, the constant is approximately equal to $\sqrt{\log_e N}$. A variation is defined as the magnitude of the change from a maximum value to the succeeding minimum value.

For design purposes, a statistical estimate of the extreme value of the various variables may be made. Let the value of E corresponding to the most severe condition expected be E. If the ship is expected to experience N variations during the time it is exposed to this

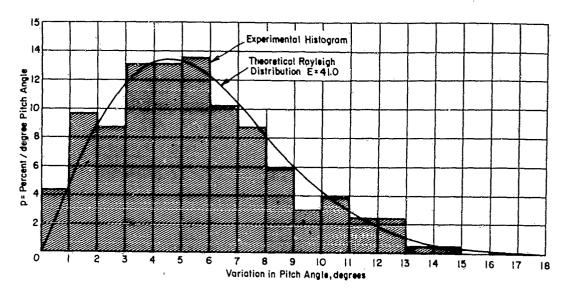


Figure 1 - Distribution of Variation in Pitch Angle for USS SPERRY (DD 697) operating condition, then

$$x_{m_1}^2 = E_m [y + \log_e N]$$

where N is assumed large.

The value of y is a function of the risk f (selected by the designer) which is the fraction of all samples of size N, belonging to a distribution specified by p(x), which will have at least one value of $x > x_{m-1}$. Table 1 of Reference 6 gives y as a function of f. For example, if one chance in a thousand is taken, f = 0.001 and y = 7.0 The value x_{m-1} is then that magnitude of the variable which, on the average, is exceeded only by the fraction f of many similar ships operating under the most severe service conditions.

AIRCRAFT CARRIERS

Rigid-body motions for ESSEX (CVA 9) Class aircraft carriers were obtained from tests conducted on USS VALLEY FORGE (CVS 45) during the period September 1955 to February 1957 in the North Atlantic, and on USS ESSEX (CVA 9) and USS ORISKANY (CVA 34) during trips around Cape Horn in the (local) winter seasons of 1957 and 1952, respectively. Data for ESSEX-Class carriers are given in References 7 and 8.

Long-Term Distributions

Figures 2 through 4 give the probability, for an ESSEX-Class aircraft carrier, of exceeding or not exceeding a given value of motion over a number of years. For example, only 1 percent of the variations in pitch angle would, on the average, exceed a value of 4 deg peak to peak; see Figure 2.

As pointed out by Jasper, 3 the usefulness of the long-term distributions does not extend

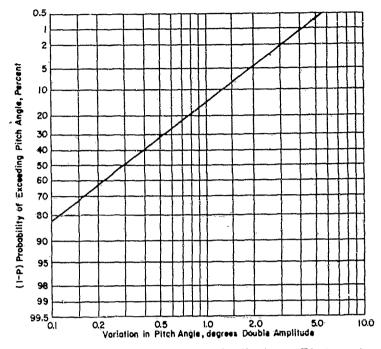


Figure 2 - Long-Term Cumulative Distribution of Pitch Angle for ESSEX-Class Aircraft Carriers

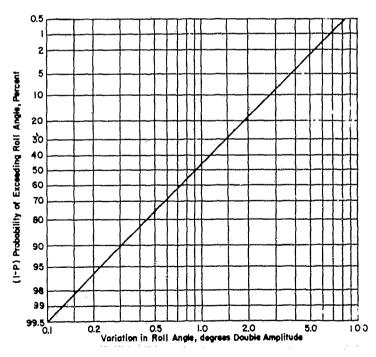


Figure 3 - Long-Term Cumulative Distribution of Roll Angle for ESSEX-Class Aircraft Carriers

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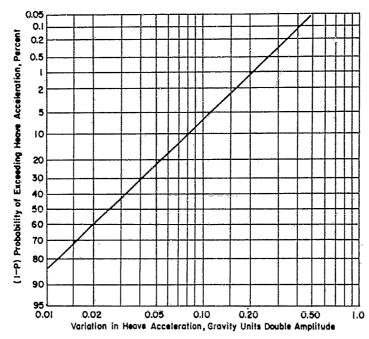


Figure 4 - Long-Term Cumulative Distribution of Heave Acceleration for ESSEX-Class Aircraft Carriers

to prediction of extreme values. The long-term distributions can probably be considered valid up to variations of 4 deg in pitch, 6.8 deg in roll, and 0.20 g in heave acceleration (corresponding to 1 - P = 0.01).

Distribution for Most Severe Operating Conditions

The maximum values of E that have been computed from experimental data on ESSEX-Class carriers for the most severe conditions encountered during the tests are 12.8 (deg)² for pitch, 61 (deg)² for roll, and 0.014 (g)² for heave. The actual operating conditions associated with these maximum values of E are listed in Table 1. From these values and Figure 5, the probability of exceeding or not exceeding a given value of ship motion under these most severe conditions can be obtained. As an example, consider the probability of exceeding a roll angle of 15 deg under the most severe operating conditions observed:

$$x^2/E = (15)^2 / 61.0 = 3.7$$

From Figure 5

(1-P)=2.5 percent.

TABLE 1

Maximum Ship Motions for ESSEX-Class Aircraft Carriers

All values are peak-to-peak variations. Data were taken from Table 7 of Reference 7.

İ		Conditions for wi	nich Extreme Valu	e is Prec	icted	Number of	Estimated	Maximum		
Quantity	Sea State	Characteristic Wave Height ft	Direction of Sea Relative to Ship Course degree	Ship Speed knots	E _{max}	Variations in 4 Hours for Steady Conditions	Most Probable Largest Variation in 4 Hours	Variation lot Life	Largest Measured Variation	Maximum Variation for Design Purposes
isaye Acceleration Rell Angle Pitch Angle Rolling Acceleration	6	20	0	10	0,014 (g) ²	1350	0.32 g	0.43 g	0,30 g	0.50 g
Rell Angle	6	20	45	10	$61.0 (deg)^2$	980	19,6 deg	32.0 deg	19.0 deg	32 deg
Pitch-Angle	6	24	45	8	12.8 (deg) ²	1180	9.5 deg	14.5 dog	9,5 deg	15 deg
Pitch-Angle Rolling Acceleration	-	20	45	10		980	0.067 rad/sec2	0,10 rad/sec		0.10 rad/sec
Pitching Acceleration*	6	24	45	8	Dec 401	1180	0.07 rad/sec ²	0.10 rad/sec ²		0.10 rad/sec2

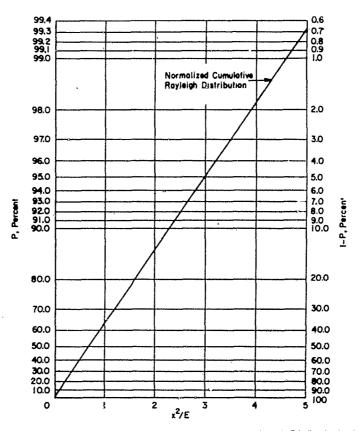
*Angular socciousies for pitching is not sinuscidal. Peak values were computed from measured displacement, data assuming the maximum value to be 1.5 times the maximum value of a single sinuscidal metion with the same and from each to the sinuscidal metion with the same and from each to the sinuscidal metion with the same and in the same and


Figure 5 - Short-Term Cumulative Distribution of Variation in Pitch Angle, Roll Angle, and Heave Acceleration

Extreme Values

The maximum predicted ship motions for an ESSEX-Class aircraft carrier computed by statistical methods are listed in Table 1. Table 1 gives the maximum motions that can be expected during any 4-hr period when operating conditions are the most severe expected for the motion under consideration, and also an estimate of the maximum motions that may be expected during the useful life of the ship. For the purpose of this report, the useful life of a ship was taken as 20 years, and a 4-hr period at the most severe operating condition was assumed to occur on an average of once every year.

The extreme values in Table 1 are based on the assumption that all variations are independent, random values. This is, in fact, not the case because consecutive motions may be shown to be partially correlated. Analysis by means of autocorrelation functions indicates that roughly every fourth cycle of motion may be considered independent. In practice this means that the predicted extreme values are conservative; i.e., they are likely to be somewhat too high. Actually these predictions of extreme values should be regarded as rough estimates.

DESTROYERS

The rigid-body motions for SUMNER (DD 692) Class destroyers were obtained from tests on USS CHARLES S. SPERRY (DD 697) during the period from September 1955 to February 1957 in the North Atlantic. 9

Long-Term Distributions

Figures 6 through 8 give the probability of exceeding or not exceeding a given value of motion over a number of years. For example, only 1 percent of the variations in pitch angle would, on the average, exceed a value of 9.5 deg; see Figure 6.

The long-term distributions can probably be considered valid up to variations of 9.5 deg in pitch, 39 deg in roll, and 0.55 g in heave acceleration (corresponding to 1-P=0.01).

Distribution For Most Severe Operating Conditions

The maximum values of E that have been computed from measurements made during sea tests of a SUMNER-Class destroyer are 41 (deg)² for pitch, 459.2 (deg)² for roll, and 0.096 (g)² for heave. Actual operating conditions associated with these maximum values of E are listed in Table 2. From these values and Figure 5, the probability of exceeding or not exceeding a given value of ship motion under these most severe conditions can be obtained. As an example, consider the probability of exceeding a roll angle of 40 deg under the most severe operating conditions observed:

$$\frac{\alpha^2}{E} = \frac{(40)^2}{459.2} = 3.49$$
, and

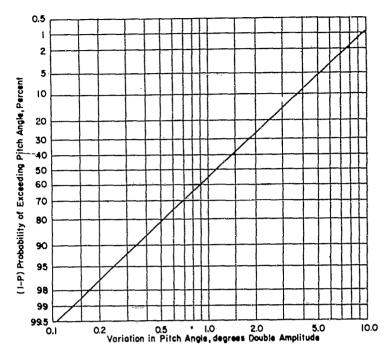


Figure 6 - Long-Term Cumulative Distribution of Pitch Angle for SUMNER-Class Destroyers

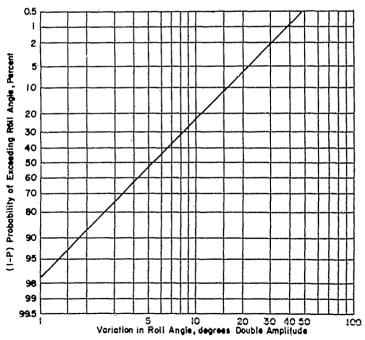


Figure 7 - Long-Term Cumulative Distribution of Roll Angle for SUMNER-Class Destroyers

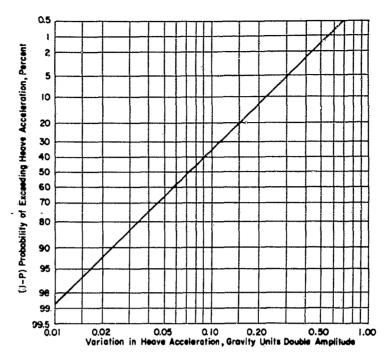


Figure 8 - Long-Term Cumulative Distribution of Heave Acceleration for SUMNER-Class Destroyers

TABLE 2

Maximum Ship Motions for SUMNER-Class Destroyers

Į		Conditions for w	hich Extreme Volu	e is Pro	dicted	Number of	Estimated	Mas in we		
Quanti ty	Sea State	Characteristic Wave Height ft	Direction of See Relative to Ship Course degree	Ship Speed knots	E _{n.est}	Variations in 4 Hours for Steady Conditions	Most Probable Largest Variations in 4 Hours	Variation for Life of Ship (/=0,001)	Largest Measured Vadation	Maximum Vipiotion for Doolen Purposes
Heave Acceleration	4	8.0	45	25	0.0%(g)2	2890	0.274 g	1.39 g	0.72	1.40 g
Rell Angle	5	10.0	confused Sea	1	450,2 (40g) ²	1616	58.2 dog	95 deg	93,8 deg	% deg
Pitch Angle	5	10.0	90	7	41.0°(40g)2	1485	17.3 dog	28 deg	15.6 dag	28 deg
Rolling Acceleration*	5	10.5	Confused Sea	i		1616	8.48 rad/sec ²	1,20rad/sec ²		0,00 md/sec2
Pitching Acceleration*	5	10.5	96	7		1485	0.19 rad/sec ²	0.30 rad/sec ²		0.30 red/sec ²

^{*}Angular acceleration for pitching is not sixusoidal. Post values were computed from measured dispinarment data, assuming the maximum value to be 1.5 times the maximum value of a playele sizusoidal notion with the same angularde and frequency.

from Figure 5

(1-P) = 3.0 percent

Extreme Values

Table 2 gives the maximum predicted ship motions for a SUMNER-Class destroyer operating in the North Atlantic Ocean under wartime conditions. It lists the maximum value of motion expected to occur during a 4-hr period when operating conditions are the most severe expected for the motion under consideration and gives an estimate of the maximum motion expected to occur during the life of the ship as before, useful life was taken as 20 years, and a 4-hr period at the most severe operating condition was assumed to occur on the average of once every year.

As in the case of aircraft carriers, the predicted extreme values may be somewhat high because of the assumption that all variations are independent.

VIBRATIONS

This section presents environmental vibration data at the service platform of radar masts. The data may be used to establish design specifications of environmental vibration for shipboard radars. These vibrations may be specified in terms of the angular and linear motions together with the associated frequencies.

The vibration at the radar platform may be considered to have the following major components:

- 1. Vibration due to the hydrodynamic forces generated by the propeller. These oscillations are predominantly at propeller-blade frequency (shaft rpm times the number of propeller blades).
- 2. Vibration due to mass unbalance of the propeller and propulsion shafting. These oscillations are at shaft frequency (shaft rpm).
- 3. Transient vibrations of the radar-mast structure corresponding to its own natural frequencies, induced by ship maneuvers, such as turns and crash-ahead crash-astern operations.
- 4. Severe whipping vibrations of the ship and the mast structure induced by slamming of the ship in heavy seas.

The amplitudes of blade-frequency and transient vibrations, Items (1) and (3), were measured largely during smooth-water operations. The values so obtained must be increased to allow for rough-water operation, gunfire, and maneuvers. The limited data available indicate that the multiplication factors given in Table 8 are reasonable. The contributions of Items (1) through (4) are added linearly to arrive at maximum values for design purposes.

TABLE 3

Factors for Converting Vibration Amplitudes in Calm Seas to Extreme Conditions

The vibration amplitude for calm-sea operation is taken as A.

		Mult	iplicat	ion F	actor		Vibra Ampli	
Type of Vibration	Rough	Seas	Shi Maneu		Gunf	ire	for Ex Condi	treme
	CVA	DD	CVA	DD	CVA	DD	CVA	DD
Propeller-excited	2	2	2	2	1	1	4A	4A
Excited by unbalance of propeller-shaft system	1	1	1	1	1	1	Α	A
Transient vibration during maneuvers	2	2	1	1	1	2	~2A	4A
Whipping vibration due to slamming	1	1	1	ì	1	1	Á	A

Data for aircraft carriers were obtained from tests on USS MIDWAY (CVA 41), ¹⁰
USS FRANKLIN D. ROOSEVELT (CVA 42), ¹¹ and ESSEX. ⁷ Data for destroyers were obtained from tests on USS GREENE (DD 711), ¹² USS DECATUR (DD 986), ¹³ and SPERRY, ⁹ and from Reference 14.

All amplitudes of vibration are given in terms of single amplitudes, as is customary. This differs from the values for the rigid-body motion given previously which are all in terms of total variations; i.e., peak-to-peak values. The values for the environmental conditions at the radar platform derived from the rigid-body motions are given in single amplitudes. The reference coordinate system is assumed to be fixed in and move with the ship. The coordinate directions are always referred to as vertical, longitudinal, and athwartship directions, even though the axes may rotate.

AIRCRAFT CARRIERS

Maximum Vibration at Radar Locations

The maximum amplitudes of vibration and the corresponding frequencies recorded on the radar masts of aircraft carriers are listed in Tables 4 and 5. Vibrations due to unbalance in propulsion system were negligibly small; hence they are omitted. From Tables 4 and 5 the conditions which give the largest accelerations in the vertical, longitudinal, and athwartship directions are selected. This gives the following probable maximum values, making due allowance for the factors listed in Table 3.

Table 4 - Maximum Vibrations Measured on Radar Masts of MIDWAY-Class Aircraft Carriers during Steady-Speed Operation in Calm Seas.

					MIDMAY	(CVA 41)		RO	FRANKLIN POSEVELT (
Type of Mast	Location	Direction of	Fo	ur-Bladed Pro	pellers	The	ee-Bladed Pi	opellers	Th	ree-Bladed P	ropellers
Type or mast	of Mast	Vibration	Shaft Speed spm	Single Amplitudes mils	Frequency cps	Shaft Speed Ipm	Single Amplitudes mils	Frequency cps	Shaft Speed rgm	Single Amplitudes mils	Frequency Cps
(SR-3 Antenna) Tripod Mast (SX Antenna) Pole Mast F	Frame 98	Vertical Longitudinal Athwartship	200 180 200	0.3 75.0 45.5	25.0 2.8 3.4	190 200 200	2.1 134.0 133.5	9,5** 2,5* 3,3	_ 155 _	14.1	- 1,7**
	Frame 105	Vertical Longitudinal Athwartship	120 180 170	5.9 15.0 5.5	8.0** 3.0 3.0	200 150 150	1.5 56.4 42.8	10.0** 2.5 2.6*	165 185 95	3,8 1,4 5,8	7,7°° 10,6°° 5,3°°
Pole Mast (SR-2 Antenna)	Frame 125	Vertical Longitudinal Athwartship	140 190 160	1.8 24.5 13.4	5.7° 12.5° 6.1°	190 200 200	29.9 29.5 31.9	9,5** 5,0* 6,0*	115	25.0	- 6,0••

^{**}This vibration is believed to correspond to propeller-blade frequency.

Table 5 - Maximum Vibrations Measured on Radar Masts of MIDWAY-Class Aircraft Carriers during Maneuvers in Calm Seas

	1 .				MIDW	Y (CVA 41)			FRANKLIN	D. ROOSEVE	LT (CVA4
	Location	Direction of	Four-B	laded Propeli	legs	Three-B	laded Propel	ers	Three-	Bladed Prope	llers
Type of Mast	of Mast	Vibration	Type of Maneuver	Single Amplitudes mils	Frequency cps	Type of Maneuver	Single Amplitudes mils	Frequency cps	Type of Maneuver	Single Amplitudes mils	Frequenc Cps
		Vertical	Crash whead from full speed astern	0.8	6.0	Hard turn to starboard 30 deg to 60 deg	4.4	9.5**	-	_	-
Pole Mast ,5R-3 ны! эпаў	Frame 98	Longitudinal	Crash ahead from full speed astern	335.0	3.3*	Hard turn to starboard 30 deg to 60 deg	169,8	2.7*	-	-	-
		Athwartship	Crash ahead from full speed astern	192,5	2,7*	-	-	-	15 deg right rudder at 30 knots	27.5	9,1**
		Vertical	Crash astern from full speed ahead	3.0	5.8	Hard turn to starboard 60 deg to 90 deg	3.0	9.5**	Coash back to full power	11.6	4.5*
Tripod Mast (SX Antenna)	Frame 105	Longitudinal	Crash ahead from full speed astern	143.0	2.8*	Hard tum to starboard 120 deg	101.6	2,4*	Crask back to full power	12,2	5,0
		Athwartship	Crash ahead from full speed aslern	192:5	2,7*	Hard turn to starboard 120 deg	104.5	2.5*	to fall power	39,3	5.0
Pole Mast	F	Vertical	Hard turn 60 dag to port	19.5	10.0**	Hard turn to starboard 180 deg	4,3	9,000	-	_	-
(SR-2 Antenna)	Frame 125	Longitudinal	Crash ahead from full speed astern	103.5	4.4*	Hard turn to port 120 deg	18,3	8.7**	-	-	-
		Athwartship	Crash astern from full speed ahead	55,0	5,5	Hard turn to starboard 180 deg	17.5	9 ,0••	Crash back In shallow water	130,00	\$.90

^{**}This ribration is believed to correspond to propeller-blade frequency.

Maximum propeller-blade-excited vibration:

Vertical direction; $29.9 \times 4 = 120$ mils at 9.5 cps

= 1.08 g

Longitudinal direction; $14.1 \times 4 = 56$ mils at 7.7 cps

= 0.33 g

Athwartship direction: $8 \times 4 = 32$ mils at 12 cps

= 0.46 g

Maximum transient vibration at the natural frequency of structure during maneuvers:

Vertical direction; $11.6 \times 2 = 23$ mils at 4.5 cps

= 0.47 g

Longitudinal direction; $335 \times 2 = 670$ mils at 3.3 cps

= 0.73 g

Athwartship direction; $180 \times 2 = 260$ mils at 5.9 cps

= 0.90 g

The maximum transient vibration due to slamming is more difficult to evaluate. When the ship slams in rough seas, the hull responds by vibrating in its flexural modes of vibration. The predominant response is in the first two modes of vertical and horizontal flexural vibration.

M les of deformation calculated for ESSEX by the method of Reference 15 are plotted in Figure 9. The difference in slope between various longitudinal locations along the ship (which may give rise to radar errors) will depend on the amplitude of vibration in the several modes. From the relative amplitudes and the maximum whipping motions expected in rough seas for ESSEX-Class carriers, the angular and linear vibratory motions at the base of the mast may be calculated.

Measurements made on ESSEX gave a maximum vertical deflection of 1.9 in. amidships corresponding to the two-noded vertical mode of vibration with a frequency of 0.8 cps. For the worst sea conditions it is estimated that a vertical deflection of $1.9 \times 1.5 = 2.85$ in. amidship, in this mode, may occur. The whipping motion for the second vertical mode has been estimated from the response in the first mode by assuming that the whipping is induced by a step load at the bow. Whipping in the vertical modes of vibration results in motions at the mast top in the vertical and longitudinal directions and rotation about an athwartship axis.

Whipping motions of the ship in the athwartship direction also occur. The limited measurements obtained during the vibration tests on ESSEX indicate that the maximum expected athwartship displacement at the stern of the ship is about 1 in. in the two-noded mode. This produces motions at the mast top in the athwartship direction, together with a rotation about the axis of the mast.

The motion of the hull due to whipping in the vertical and athwartship directions at selected locations and the corresponding angles of rotation are listed in Table 6. The

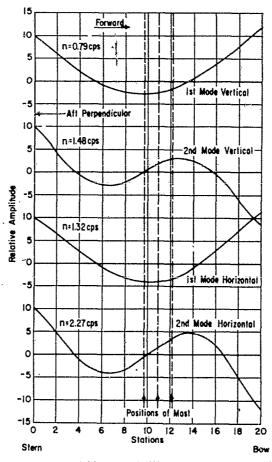


Figure 9 - Normal Modes of Vibration for ESSEX-Class
Aircraft Carriers Calculated on UNIVAC
Displacement = 38,300 tons.

motions at the top of the mast positioned at the selected locations are listed in Table 7. For the calculations, the mast was assumed to be rigid and to extend 166.4 ft above the center of gravity of the ship; see Figure 10. Maximum accelerations resulting from propeller blade, transient, and whipping excitations are given in Table 8.

Elastic Distortion between Radar and Its Reference Gyro

There will, in general, be angular distortions between the location of the radar and the reference gyro. These differences in angle may produce radar errors. The differences arise from two sources: the elastic deformation of the hull, and the elastic deformation within the radar mast.

The maximum relative angular motion due to hull vibration will be computed on the assumption that both the mast and the stable element are located within one-half a ship length

TABLE 6

Vertical and Athwartship Whipping Motions of Carriers Due to Slamming

		Vertical '	Whipping			Athwartshi	p Whipping	
Longitudinal	First N 0.79 c		Second 1.48		First & 1.32 (Second 2,27	
Location of Mast Stern Mast at Forward End of Island Mast at Center	Amplitude	Angle radians	Amplitude	Angle radians	Amplitude in.	Angle	Amplitude	Angle radians
	in.	× 10 ⁻³	in.	× 10 ⁻³		× 10 ⁻³	in.	$\times 10^{-3}$
Stern	9.5	3.85	3.7	2.2	1.0	2.5	0.32	1,32
Mast at Forward End of Island	1.43	2.70	1,13	0.35	0.3	0.16	0.10	0.12
Mast at Center of Island	2.38	1.32	0.75	1.3	0.6	0.07	0.04	0.12
Mast at Aft End of Island	2.85	0	0.19	1.3	0.4	0.12	0.03	80.0

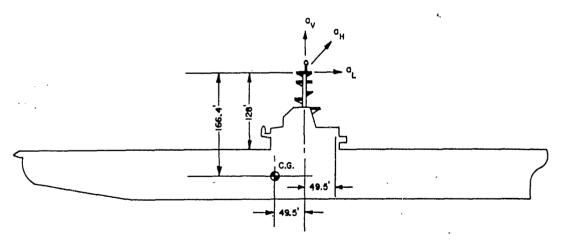


Figure 10 - Inboard Profile for ESSEX-Class Aircraft Carriers

Table 7 - Displacements and Accelerations at Radar Service Platform on Carriers Due to Slamming (Whipping Response)

					Linear Motie	Linear Metion at Radar Service Platform	ice Platferm					Rotation	Rotational Motion at Radar Service Platform	Radar Servi	ce Platform	
in the second se	Ž		Vertical			Athwartship			Lengitudinal			About Vertical Axis			About Athwartship Axis	<u>.</u>
Location of Wast Wast	Vibrition	Amplitude	Acceleration	Frequency	Amplitude	Acceleration	Frequency Amplitude	Amplitude	Accelemtion	T.	Angle radians	Acceleration radian per sec ²	Frequency	Angle : radians	Acceleration radian per sec ²	Frequency
		. <u>.</u>	_	5	<u></u>	-	<u>.</u>	Ė	*	3	s-91 ×	× 10 ⁻²		× 10-3	× 10-2	
Ferward End		3.	8.0	3	2	0.05	1.32	5.4	0.35	70	0.16	1.10	1.32	2.7	0′2	80
ef Island	Second	=======================================	0.25	₽.	9.9	0.05	2.23	7.0	0.15	1.48	0.12	2.45	227	135	3,0	1.48
Center of		2,38	0.15		9.0	0.13	1.32	2,65	0.17	8.0	0.07	050	132	132	3,4	T 0
Islam		K.0	6.17	1.4	3.	0.02	2.27	2.50	0.57	1.48	0.12	2.4	171	2	 	1,4
AR End of		2.85	9.15	6.3	0.40	0.07	1.32	•	•	70	9.12	97	13%	-	-	70
Island		9,19	٠ <u>.</u>	1.4	0.03	20.0	2.27	2.60	0.57	1.48	10.	12	227	1.30	11.3	1.48

Table 8 - Maximum Vibrational Accelerations for Carriers at Radar Service Platform

				Kaxi mu	Maximum Linear Acceleration in g	eration in g					Haximum Angular Acceleration racion per sec ²	رم الله الله الله الله الله الله الله الله
Vibration		Mast at Fervard End of Island	ž ,		Mast at Center of Island	*		Mast at Aft End of Island	7	ista	Hast at Cooks of Island, About Indicated Axis	n el sediAxis
	Vertical	Athwartship	Vertical Admuniship Longitudinal Vertical Attruntiship Longitudinal Vertical Admuniship Longitudinal	Vertical	Athwartship	Longitudinal	Vertical	Aftwartship	Longitudinal	I	Vertical Athwastable Longifiedina	Longifüdinai
Prepetter	8.	0.46	0.33	1.06	0.46	0.33	1.01	37.0	.633		(30 ()	0.118
Maneuver	9.0	9.9	6.73	9.05	8.0	0.73	50.0	0.00	0.73	•	0.185	0.025
Shaft	5	•	•			ó	•	-	•	•	•	•
Whipping, First Made	6	9.05	6.35	0.15	9.10	0.17	9.18	100	•	564.0	2034	•
Whipping, Second Mode	0.23	50.0	0.15	0.17	0.02	0.57	10.0	9,02	15.	\$2 6.0	1111	0
Tetal Acceleration	171	1.46	\$.1	1.45	7:	8.1	1,58	1.45	1.63	200	8176	0.143

aft of the assumed location of the mast. The computed maximum angular distortions within this length, about an athwartship axis and about the axis of the mast located at the center of the island, are listed in Table 9. The maximum angular rotations of the radar platform relative to the base of the mast were computed on the assumption that these angles are equal to the maximum amplitude of mast vibration divided by the height of the radar platform above the flight deck (128 ft). The sum of these angular components is assumed to represent the most severe, expected, angular error between the radar and its reference gyro. Table 9 gives the resulting values.

TABLE 9

Elastic Angular Distortion on Carriers between Radar Platform and Reference Gyro for Mast Position at Center of Island

Relative Angular		ntribution from ortion, radians			on from Mast Idians× 10 ⁻³	Total Angular
Distortion about Indicated Axis	Vertical Vibration	Athwartship Vibration	Torsional Vibration	Athwartship Vibration	Longitudinal Vibration	Distortion radians× 10 ⁻³
Vertical	Ú	2,50	0	0	0	2.50
Athwartship	81.7	0	0	0	0.47	9.17
Longitudinal	Ú	0	0	0.19	0	0.19

DESTROYERS

Maximum Vibration at Radar Locations

The maximum amplitudes of vibration and the corresponding frequencies recorded on the radar tripod mast on destroyers of SUMNER¹² and FORREST SHERMAN (DD 931)¹³ Classes are listed in Table 10. From Table 10 and Reference 14 the conditions which give the largest accelerations in the vertical, longitudinal, and athwartship directions are selected. This gives the following maximum values, making due allowance for the factors given in Table 3.

Maximum propeller-blade-excited vibration:

The propeller-blade-excited vibrations at the mast top will be the same order of magnitude as those normally experienced by the hull at Frame 72. These magnitudes for the longitudinal and athwartship directions were obtained from Table 1 of Reference 14.

Vertical direction; $6.0 \times 4 = 24.0$ mils at 13 cps

= 0.41 g

Longitudinal direction; $24 \times 4 = 96$ mile at 7.5 cps

= 0.54 g

Athwartship direction; $15 \times 4 = 60$ mils at 10 cps

= 0.6 g

TABLE 10

Maximum Vibrations Measured on Platform of Tripod Radar Masts on Destroyers during Steady-Speed Operation and Maneuvers

Destroyer Class	Shaft Speed rpm	Direction of Vibration	Single Amplitude mils	Frequency cps	Type of Operation	Remarks
	260	Vertical	6.0	13.0	Steady runs	_
SUMNER	330 150	Lengitudinal	27.0 24.0	6.0 7.5	Steady runs Steady runs	- -
	200	Athwartship	15.0	10.0	Steady runs	_
	310 210 310	Vertical	4.0 8.0 12.5	6.5 2.4 6.0	Steady runs Steady runs Maneuvers	- - -
FORREST SHERMAN	310 160 310	Longitudinal	30.0 70.0 65.0	5.2 1.4 5.8	Steady runs Steady runs Maneuver	Shaft frequency Shaft frequency Natural frequency of mast
	310 200 310	Athwartship	28.0 32.0 37.0	5.2 3.3 6.3	Steady runs Steady runs Maneuver	Shaft frequency Shaft frequency Natural frequency of mast

Maximum transient vibration at the natural frequencies of the mast during maneuvers:

Vertical direction;

 $12.5 \times 4 = 50.0$ mils at 6.0 cps

= 0.18 g

Longitudinal direction;

 $65 \times 4 = 260$ mils at 5.8 cps

= 0.89 g

Athwartship direction;

 $37 \times 4 = 148$ mils at 6.3 cps

= 0.59 g

Maximum vibration excited by shaft unbalance:

Vertical direction;

very small.

Longitudinal direction;

 $30 \times 1 = 30$ mils at 5.2 cps

 $\approx 0.081 g$

Athwartship direction;

 $28 \times 1 = 28$ mils at 5.2 cps

= 0.076 g

Only few data are available for the first-mode vertical and athwartship flexural vibrations of a destroyer due to slamming. These values were obtained on SPERRY in April 1956 during operation in a State 6 sea superimposed on a swell. The direction of the seas was 45 deg relative to ship course. These values are used for estimation of the response in the corresponding second modes in the same way as used for carriers. The cal— And mode shapes

are shown in Figure 11, and the inboard profile for SUMNER-Class destroyers with the necessary dimensions is shown in Figure 12. The vibrations of the hull and of the radar service platform resulting from slamming calculated for the position of the radar mast shown in Figure 12 are listed in Table 11..

The maximum linear accelerations at the location of the radar in three orthogonal directions and the three maximum angular vibrations are listed in Table 12.

Elastic Distortion Between Radar and its Reference Gyro

The maximum angular distortion of the radar platform relative to its gyro is computed on the assumption that the gyro position is located one-half a ship length aft of the mast. The total angular distortion, the sum of hull and mast distortion, is listed in Table 13.

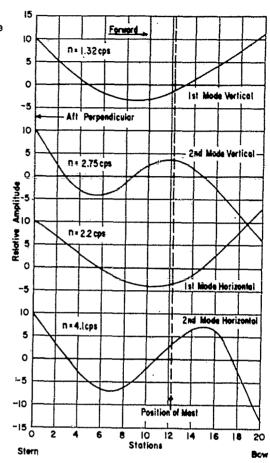


Figure 11 — Normal Modes of Vibration for SUMNER-Class Dostroyers from Vibration Generator Tests

Displacement - 3265 - 3220 tons

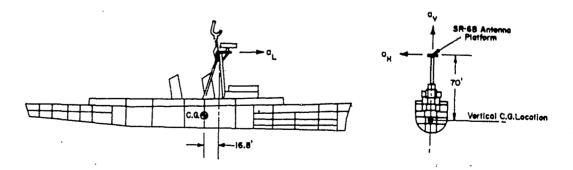


Figure 12 - Inboard Profile for SUMNER-Class Destroyers

Table 11 Itali and Radar Mast Whipping Vibration for Destroyers Due to Slamming

			Ku	II Whipp	Hull Whipping Melion	£							4	Motion	Motion at-Radar Service Platform	Service	Platfor		·				
·		Vertical	[C3]			Athwartship	rtship					5	Linear						Rotation About Indicated Axis	About	Indicate	d Axis	
Longitudinal	First Mode 1,32 cps		Second Mode 2.75 cps	Mode		First Mode 2.2 cps	Second Mode 4.1 cps	Mode		Vertical		¥	Athwartship		Ľ	Longitudinas		>	Vertical		¥	Athwartship	
Lecation on Ship	•butilqmA in,	Angle (adjans × 10 ⁻³	Amplifude in.	ϵ -01 \times ensiber	AbufilqmA fn.	signa $^{6-01}$ × snaibar	Amplitude in.	Angle $^{5-01} \times ^{20-3}$	ebutilqaA .ni	Acceleration T	bz _tedneuck	Amplitude in,	noitatelessA	cba Liedneuck	ebutliqmA In.	tcceletation .	ba (edneuc)	e-Ol × ansiba	C=01 × Epse/be	:bz _tedneuck	s-01 × snsibs	sd/sec ² × 10 ⁻²	cba Liedneucy
\top	3.3	3.32	0.75	1.23		0.376	0.15	0.226				,		1				,	¥		1	1	
Point 16.8 It find of C.G.	623	2.04	0.26	0.085	0.17	0.141	0.045	0.168															
Mast top 70 ft frem									0.33	90.0	1.32	0.17	0.084	22	17.1	0.3	1.32	17.0	2.69	2.2	2.04	13.5	132
C.G. 16.8 It fwc ef C.G.									0.26	0.20	2.75	0.045	7.000	7	0.07	900	2.75	0.168	11.5	3	380'0	2.53	2.75

TABLE 12

Maximum Vibrational Acceleration for Destroyers at Radar Service Platform

Cause of Vibration	Maximum Linear Acceleration in g					
oduse of Vibration	Vertical	Athwartship	Longitudinal			
Propeller	0.41	0.6	0.54			
Maneuver	0.18	0.59	0.89			
Shaft	-	0.08	0.08			
Whipping, First Mode	0.06	0.08	0.30			
Whipping, Second Mode	0.20	0.08	0.06			
Total	0.85	1.43	1,87			
		num Angular Aco bout Axis in rad				
Propeller	_	0.31	0.33			
Maneuver	- 1	0.48	0.32			
Shaft	-	0.05	0.04			
Whipping, First Mode	0.03	0.14	_			
Whipping, Second Mode	0.12	0.03				
Total	0.15	1.01	0.69			

TABLE 13

Elastic Distortion between Radar and Reference Gyro for Destroyers

The values given in this table are derived from the summation of the contributions from the first and second modes of flexural hull vibration.

Relative Angular		Contribution fr Hull Distortio	n,	Mast D	ition from istortion,	Total Angular Distortion
Distortion		radians× 10 ⁻	3	radians	$\times 10^{-3}$	• • • • • • • • • • • • • • • • • • • •
about Axis	Vertical	Athwartship	Torsional	Athwartship	Longitudinal	radians× 10 ^{−3}
Vertical	0	0.91	0	0	0	0.91
Athwartship	6.92	0	0	0 .	0.54	7.46
Longitudinal	0	0	Negl.	0.33	0	0.33

APPLICATION TO DESIGN

DETERMINATION OF MAXIMUM ACCELERATIONS AT RADAR SERVICE PLATFORM

The extreme motions at the radar service platform on carriers and destroyers during operation in heavy seas may be used as a guide for structural design of radar antennas or for specifications of stabilization requirements. The following outline is intended only as an indication of possible procedures.

The maximum accelerations at the radar service platform in three linear and three rotational directions depend on the rigid-body motion and the elastic vibration of the ship. The maximum values due to the elastic vibrations for carriers and destroyers have been given in Tables 8 and 12, respectively. It is, therefore, necessary to add to these values those due to the rigid-body motions in rough seas. The amplitude of the rigid-body motions will be taken as one-half the magnitude of the "variation." Although maximum values may not occur simultaneously, they may conservatively be assumed to do so for purpose of design. The calculations have been made for the estimated, most probable, largest variation during a 4-hr operation and for the maximum variation for design purposes. The respective values of pitch, roll, and heave are listed in Tables 1 and 2 for ESSEX-Class carriers and SUMNER-Class destroyers, respectively. The ship mations are referred to orthogonal axes through the center of gravity which are fixed in the ship, and the coordinate directions are denoted vertical, longitudinal and athwartship directions.

The accelerations at the radar service platform due to the rigid-body motions depend on the location of the platform relative to the ship's center of gravity. Typical dimensions are shown in Figures 10 and 12. The calculations for the radar service platform for carriers will be made on the assumption that the radar mast is located at the center of the island.

The maximum heave acceleration is taken in the vertical direction. The rolling acceleration produces linear components in the vertical and athwartship directions, and the pitching acceleration produces linear components in the vertical and longitudinal directions. If h is the height of the mast above the center of gravity, ω_r the rolling frequency in radians per second, and ϕ_r the maximum angle of roll in radians, the accompanying vertical acceleration is $h w_r^2 \phi_r^2$ and the athwartship acceleration is $-h w_r^2 \phi_r$. These maxima do not occur at the same time; the maximum vertical acceleration occurs at zero angle and the maximum athwartship acceleration at maximum angle.

The accelerations due to pitching result in two terms for each direction. The vertical acceleration is

$$h \omega_p^2 \phi_p^2 \cos^2 \omega_p t - d \omega_p^2 \phi_p \sin \omega_p t$$

where ω_{p} is the pitching frequency in radians per second,

 ϕ_p is the maximum angle of pitch in radians,

d is the distance from the center of gravity of the ship to the base of the mast, and

t is the time.

The acceleration in the longitudinal direction is

$$-\hbar \omega_p^2 \phi_p \sin \omega_p t - d \omega_p^2 \phi_p \cos^2 \omega_p t$$

The sine and cosine factors are included to indicate that the maximum value in each case depends on the weight of each term.

The calculations of the maximum accelerations due to the rigid-body motions are summarized in Table 14 for carriers and in Table 15 for destroyers. The values of the linear accelerations for the most probable largest variation during a 4-hr operation in heavy seas and for the largest probable variation during the lifetime of a ship are both of the order of 2g. The vertical acceleration is superimposed on gravity. Components in the longitudinal and athwartship directions associated with gravity are automatically included. The estimated maximum angular accelerations for destroyers are considerably larger than those for carriers.

TABLE 14

Extreme Values of Accelerations Due to Rigid-Body Motions and Vibrations at Location of Radar for Carriers

All values are single amplitudes.

Source of	L	inear Accelerati	ear Accelerations, g		ular Acceleratio Axes rad/se	
Acceleration	Vertical	Athwartship	Longitudinal	Vertical	Athwartship	Longitudinal
	Case I - B	ased on most p	obable largest v	ariation duri	ng 4 <u>hr</u>	
Gravity	1.0	0	0	0	0	Ò
Heave	0.16	0	0	0	0 '	0
Roll	0	0.17	0	0	0	0,034
Pitch	0	0	0.17	0	0.035	0
Total from Vibrations, Table 8	1.45	1.48	1.80	0.029	0.413	0.143
7 101 2110110 1 1 1 1 1 1 1 1 1 1 1 1 1 1	2.61	1.65	1.97	0.029	0.448	0.177
	Case II -	Based on maxi	mum variation fo	r design pur	poses	
Gravity	1.0	0	0	0	0	0
Heave	0.25	0	0	0	0	0
Roll	0	0.28	0	Q	0	0.05
Pitch	0	0	0,27	Ō	0.05	0
Total from Vibrations, Table 8	1.45	1.48	1.80	0.029	0.413	0.143
. 12, 20, 20, 20, 20, 20, 20, 20, 20, 20, 2	2.70	1,76	2.07	0.029	0.463	0.193

TABLE 15

Extreme Values of Accelerations Due to Rigid-Body Motions and Vibrations at Location of Radar for Destroyers

All values are single amplitudes.

Source of	L	inear Accelerat	ions, g	Angular Accelerations, about Axes rad/sec ²		
Acceleration	Vertical	Athwartship	Longitudinal	Vertical	Athwartship	Longitudinal
	Case I - B	ased on most pr	obable largest v	ariation duri	ng 4 hr	
Gravity	1.0	0	0	0	0	0
Heave	0.16	0	0	0	0.	0
Roll	0	0.55	0 [0	0	0.24
Pitch	0	0	0.21	0	0.095	0
Total from Vibrations, Table 12	0.85	1.43	1.87	0.15	1,010	0.69
, , , , , , , , , , , , , , , , , , , ,	2.01	1.98	2.08	0.15	1.105	0.93
	Case II -	Based on maxi	mum variation fo	r design puŋ	osas	
Gravity	1.0	0	0	0	0	0
Heave	0.25	0	0	0	0	0
Roll	0	0.90	0	0	0	0.40
Pitch	0	0	0.33	0	0.15	0
Total from Vibrations, Table 12	0.85	1.43	1.87	0.15	1.01	0.69
interional tente se	2.10	2.33	2,20	0.15	1.16	1.09

It should be emphasized that the estimates of extreme values of accelerations that are not expected to be exceeded in heavy-sea conditions were made on the assumption that maximum vibration and rigid-body motion occur at the same time and that the radar mast is a rigid mast flexibly attached to the ship. The direct superposition of maximum values will tend to give an overestimate; the assumption of a rigid mast, on the other hand, will tend toward an underestimate.

MAXIMUM ANGULAR ERRORS DUE TO HULL AND MAST DISTORTIONS

The maximum angular distortions that may occur during ship operation in rough seas are due to elastic deformation incident to: (a) slamming, (b) hull vibration, and (c) ordinary hogging and sagging. The angular distortions given in Tables 10 and 14 include the effects of (a) and (b) only, and must be augmented to allow for (c). Data obtained on ESSEX indicate that the distortion due to hog and sag is of the order of 40 percent of the first-mode whipping response induced by slamming. The total distortion resulting from the superposition of all three effects is given in Table 16. Proper location of the reference gyro with respect to the radar mast could reduce these values considerably.

TABLE 16

Maximum Angular Elastic Distortion between Radar and Reference
Gyro for Carriers and Destroyers

Relative Angular Distortion about	Maximum Angul	ar Distortion, deg
Axis	Carriers	Destroyers
Longitudinal	0.011	0.019
Athwartship	0.59	0.55
Vertical	0.144	0.052

DETERMINATION OF STABILIZATION REQUIREMENTS

The distribution functions for the ship motions in rolling and pitching, for both the long-term and the short-term distributions, may be used to determine stabilization criteria for the ship or for the radar antenna. A design which would provide stabilization against the most severe predicted, rolling and pitching motions would be uneconomical. It seems more expedient to limit the requirements so that a certain percentage of the motions will be stabilized. The designer will have to determine the conditions for which the stabilization appears to be necessary. This may require stabilization up to a certain sea state or stabilization for a certain percentage of all motions experienced during the lifetime of the ship. Possible application will be illustrated by examples which will consider roll stabilization only; the same procedure applies for pitch stabilization.

Example 1 - A carrier may be equipped with a stabilizer designed to reduce rolling motions of the ship from 8 deg to 2 deg. What stabilization of the radar system is required for 99-percent effectiveness against all motions encountered during the most severe operating conditions?

It will be assumed that the ship stabilization of 6 deg reduces all larger roll angles by the same amount. The motions of the ship during the most severe operating conditions are specified by the short-term distribution. Measurements on ESSEX-Class carriers gave $E_{\max}=61$. From Figure 5, $\frac{x^2}{E}=4.6$ for a probability of 99 percent. From this the roll angle is calculated as $\frac{x}{2}=8.35$ deg, single amplitude. Stabilization of the radar antenna, therefore, would be required for 8.35 deg minus 6 deg, or 2.35 deg. Without stabilization of the ship, the radar antenna would require stabilization for 8.35 deg, single amplitude. The required torque for stabilization against roll depends on the angular accelerations and the mass moment of inertia of the equipment to be stabilized. The required accelerations are given in Table 14 for the unstabilized ship.

Example 2 - It is required to design a stabilizer on a destroyer that will effectively provide stabilization for a radar antenna for: (a) 95 percent of all rolling motions encountered during

the lifetime of the ship, and (b) the estimated most probable largest variation of roll in 4-hr operation in heavy seas.

The solution to (a) utilizes the long-term cumulative distribution of roll angle. The roll angle which is not expected to be exceeded by 95 percent of all rolling motions taken from Figure 7 is 22 deg; i.e., a roll angle of 11 deg single amplitude.

The most probable largest angle of roll in heavy seas (b) is listed in Table 2 as a variation of 58 deg; i.e., a single amplitude of 29 deg.

STRUCTURAL DESIGN OF MAST

Example 3 - It is required to design a mast structure for a carrier to carry a radar system. What are the loads, due to the radar, for which the mast should be designed? The radar unit has a mass M and mass moments of inertia I_x , I_y , I_x about vertical, athwartship, and longitudinal axes fixed at the center of gravity of the radar unit.

The required load may be estimated by taking the product of inertia and acceleration. The values of acceleration are taken from Table 14, Case II:

The axial load will be 2.70 M.

The athwartship load will be 1.76 M.

The longitudinal load will be 2.07 M.

The moment about the athwartship axis through the center of gravity will be 0.463 l_y ; about the mast axis 0.029 l_z ; about the longitudinal axis 0.193 l_x .

SUMMARY

- 1. Statistical methods have been used to predict probable extreme values of ship motion to be expected over the operating life of carriers and destroyers.
- 2. Graphs have been given from which the probability of exceeding any magnitude of motion may be read.
- 3. The environmental vibrations experienced at radar locations during operation in smooth seas have been given. Magnification factors are tabulated which, when applied to vibration amplitudes measured during smooth-sea operation, give an estimate of the amplitudes to be expected during operation in rough seas.
- 4. The extreme values of linear and angular accelerations expected at radar locations due to the combined rigid-body and vibratory motions have been tabulated.
- 5. I eximum angular distortions between the location of radars and the reference gyros have been computed. These distortions may give rise to inaccuracies in radar signals and thus limit radar capabilities.

6. The values given in this report refer to aircraft carriers and destroyers. They are based on reasonably well-established values for the rigid-body motions. Vibration measurements taken on ships at rough-sea conditions are scarce, and additional measurements over long operational periods and for severe sea states are needed to improve upon the numerical values given in this report. It may be reasonably expected that ships of similar design will be subjected to similar conditions during their operational lifetime. Little is known about extreme motions and vibrations of ships whose design departs considerably from the abovementioned types.

ACKNOWLEDGMENT

The authors accumulated many of the needed data from existing reports. The calculations were checked by Mr. J.F. O'Donnell, Jr. Dr. N.H. Jasper contributed enlightening discussions and helpful suggestions during the preparation of this report.

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